CONSTRUCTIONAL MODIFICATION OF AN EXTRUSION DIE

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The paper deals with design and optimization of an extrusion die which consists of a die body and a set of replaceable plugs for various extrusion cross-sections. This system is primarily used for producing WPC products on the laboratory extruder EUROTECH EB30. The paper offers several designs of extrusion dies and examines their advantages as well as disadvantages. Based on this analysis, the best design is chosen and then flow simulations through a die and by FEA are done.

Keywords

Wood, plastics, composite, WPC, extruder, die, extrusion, FEA, flow simulation

1 INTRODUCTION

One of the most widely used method for the WPC manufacturing is extrusion. Extrusion is continuous pushing of material through the tool - die. Die determines shape of final product during extrusion. For complex profiles it is very difficult to design die and design quality still depends to a large extent on the constructor's empirical data. At present there is software allowing simulation of material flow through die and therefore die designing and optimizing is faster and easier.

Our new die designs have been formed with a certain relationship to the original die, because of some structural dimensions which had to be respected.

The task was based on the previous experiences with the production of WPC on a laboratory extrusion line EUROTECH EB30. There have been several unsuccessful attempts to extrude the wood-plastic composite through the old die. All the attempts ended by extrusion of the burnt materials or by clogging the die. As the primary cause of failure we have identified the bad design of the extrusion head + the die. This whole part (the extrusion head + the die) has universal character. From the screw extruder the material does not flow directly into the die, but into the eccentric head. By rotating the eccentric, it is possible to adjust the flow of the material either to the pelletizing head or to the die for continuous extrusion.

Moreover this path to the die is leading through another "turn" (so that the material is extruded parallel to the screw axis). In this area there is an excess resistance against the material flow and therefore in our opinion this design solution is an inappropriate one.

Figure1. Extruder EUROTECH EB30
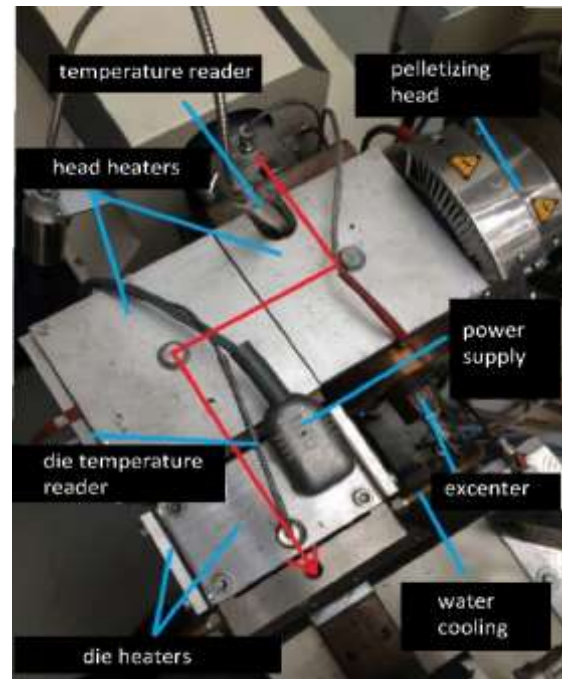
2 ORIGINAL SOLUTION

Figure 2. Original extrusion head and die

The top view of the die is shown in Figure 2. The red arrow shows material flow from the extruder into the die. As already mentioned, by placing the die right behind the extruder it is possible to avoid two turns that would significantly facilitate material flow. The picture shows also the power supplied heaters with the temperature readers. After heating to work temperature it is possible to rotate the excenter to change current flow.

All four pieces (parts) of the old die are made of the same material. The material chemical analysis has shown the die material is the most similar to ferritic steel alloyed with molybdenum X6CrMoNb17-1. Its precise

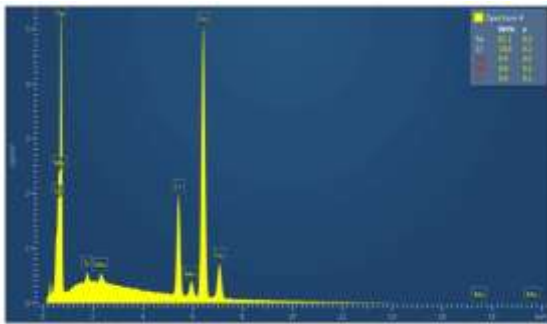chemical composition is given by the chemical analysis report.

Figure 3. Chemical composition of the original die

The material shows an exceptional resistance to corrosion, machinability, chemical resistance, polishability and exceptional mechanical properties. Based on these properties the material has been found suitable for a new die design.

3   STARTING POINTS FOR NEW SOLUTION

All general recommendations (according to professional literature) on character and die construction can be summarized in the following points [Rauwendaal 2014], [Michaeli 2003], [Kostic 2011]:

-   No "stagnation areas" in flow passage – areas where material flow is being slowed down or stopped;
-   Fluent flow rate increase along flow passage;
-   Simple installing and disassembling;
-   Length of relaxing area should be 10 times larger than diameter of flow passage- but not too long as this increases pressure gradient and decreases extrusion rate;
-   If possible prevent sudden changes in flow passage shape – if possible use small converging angles, within range: 10-30°;
-   Use large radii (either outer or inner) instead of sharp edges – the smallest radius possible: 3mm;
-   Keep the same wall thickness – for uniform temperature regulation;
-   Prevent too thick walls;
-   Construct inner walls thinner than outer walls – because of cooling;
-   Minimize using hollow areas – it is very difficult to reach high precision;
-   Die should consist of several parts (with the same width for temperature regulation if possible);
-   Flow passage axis should be symmetrical to outer walls – due to heat conduction;
-   Minimize quantity of elements in flow passage;
-   Secure minimum pressure gradient in die – through simulations;
-   The better roughness the better conditions for material flow, however maximum roughness is Ra 6.3;
-   Secure the most uniform material flow possible.

 All the above-mentioned recommendations and limitations have been included into a new die design

The old die consists of four parts. Regarding the existing heating and cooling system of the original extrusion head and die, this design of four parts has been observed in the new head and die. The main aim has been to exclude two 90° flow direction turns and to solve a requirement for die "versatility" regarding shapes of extruded profiles. To search for a solution and decide on dimensions and angles we have simulated (by Flow Simulation 2017 Solid Works Program) material flow in the old die and in several alternative solutions for the new die.

We have chosen a concept of assembled head and die, it means the joint connection consists of four parts as the original one, but the inlet channel can be changed by replaceable plug (Figure 4).

The objective of the simulation has been to search for right sizes of two parameters (radius and angle) of inlet flow passage and their mutual position with regard to the subsequent marginal conditions:

-   Flow passage inlet diameter must be d=25mm;
-   Flow passage outlet diameter must be d=5mm;
-   Minimum relaxation length in this case is 50mm;
-   Reduction angle of flow passage diameter must be 30° maximum.
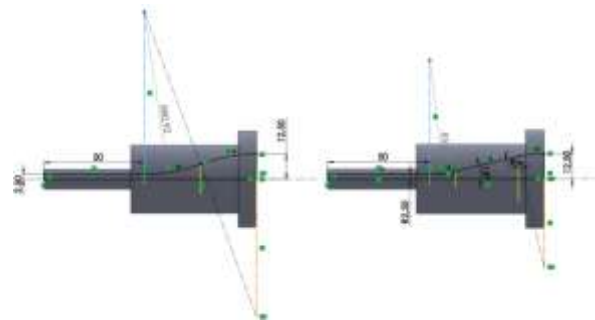

Figure 4. Inlet flow passage scheme

In total, 12 alternative solutions have been simulated based on the considered dimensions according to Table 2.

radius[mm] angle[°]	40	70	100	130	140
12	YES	YES	NO	NO	NO
13	YES	YES	YES	NO	NO
14	YES	YES	YES	NO	NO
15	YES	YES	YES	YES	NO

Table 1. Angle and radius design

Set simulation parameters must be same for all designs. The set parameters were:

-   Heat conduction in solids: ☑;
-   Time-dependent simulation: ☑;
-   Influence of gravity (Y-axis direction): ☑,
-   Solids material: stainless steel 302,
-   Wall temperature: 450K,
-   Wall roughness: 1.6 μm,
-   Outlet flow volume: 2 7475 . 10-5 m3 . s-1,
-   Atmospheric pressure and outlet temperature: 101325 Pa and 300K,
-   Mesh: automatic,
-   Flowing medium: PE.

Unfortunately, at this stage of research not all relevant WPC parameters are available to simulate under real WPC parameters. However, we assume that PE flow simulation results show suitability of our die construction modifications (Figure 6).

Table 2 shows that, in every respect, the best combination is φ=15°, r=130mm. The second-best combination is φ=13°, r=100mm.

Goal (Value)	φ=12°, r=40mm	sín., 2x r=81mm	φ=12°, r=70mm	φ=13°, r=40mm	φ=13°, r=70mm	φ=13°, r=100mm
static pressure [Pa]	33 950 632,03	36 732 236,35	34 071 291,75	36 624 269,56	35 342 082,25	33 684 122,41
absolute pressure [Pa]	33 950 645,12	36 732 250,06	34 071 304,60	36 624 283,34	35 342 095,48	33 684 135,05
dynamic pressure [Pa]	13,09	13,71	12,85	13,78	13,24	12,63
speed [m/s]	0,0698	0,0688	0,0685	0,0692	0,0686	0,0681
force [N]	16 799,69	18 172,68	16 858,08	18 119,78	17 485,52	16 666,94
forceX) [N]	-0,05	-0,39	-0,22	-0,34	-0,33	0,84
force (Y) [N]	-2,79	-2,16	-2,21	-1,85	-1,87	-2,00
force (Z) [N]	-16 799,69	-18 172,68	-16 858,08	-18 119,78	-17 485,52	-16 666,94
shear force [N]	869,09	944,07	894,57	947,53	916,98	881,89
shear force (X) [N]	123,20	131,56	131,59	132,18	131,55	131,44
shear force (Y) [N]	0,66	0,30	0,36	0,34	0,29	0,29
shear force (Z) [N]	-860,31	-934,86	-884,84	-938,26	-907,49	-872,04
shear stress [Pa]	35 229,65	38 020,22	36 003,23	38 301,78	36 914,70	35 402,72
shear stress (X) [Pa]	4 469,55	4 748,95	4 722,97	4 789,80	4 734,06	4 697,68
shear stress (Y) [Pa]	24,03	10,79	12,90	12,48	10,37	10,48
shear stress (Z) [Pa]	-31 211,38	-33 745,38	-31 757,95	-34 000,98	-32 658,09	-31 166,57
input static pressure [Pa]	34 330 311,28	37 134 846,43	34 456 558,73	37 026 721,34	35 734 739,67	34 065 466,00
output static pressure [Pa]	100 490,98	100 490,98	100 490,99	100 490,98	100 490,99	100 490,99
pressure gradient [Pa]	34 229 820,30	37 034 355,46	34 356 067,74	36 926 230,36	35 634 248,68	33 964 975,01

Table 2 Results of different combinations of angle with radii

4 CONCLUSIONS

The most important parameter of all is pressure, which course is shown on Figure 5, more precisely pressure gradient and force, more precisely its component in Z-axis. The lower pressure gradient arises in flow passage, the higher performance can be reached. Z-axis force component will be used again for inspecting screw connections in strength analysis. The graph shows that 15°angle offers the widest range of achievable pressures and it is possible to achieve the highest number of combinations.

Based on the analysis and evaluation of the simulation results new drawings of head and die with replaceable plug have been drawn up.

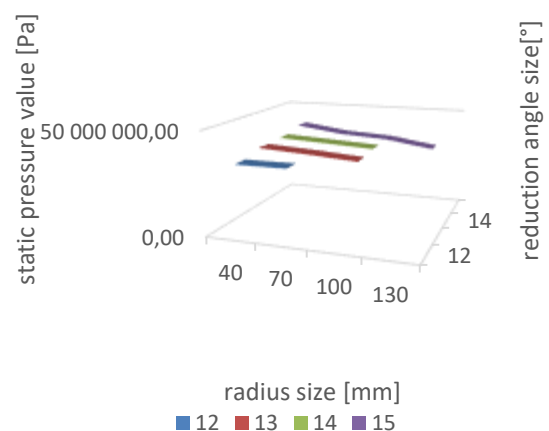Dependency of static pressure course on reduction angle and radius size

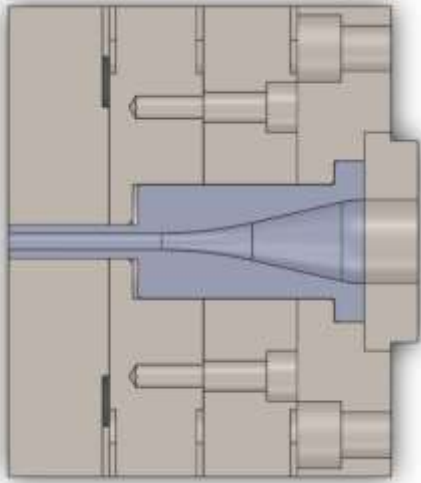**Figure 5.** Pressure dependency on angle and radius size

Figure 6. Plug placing

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