

OPERATIONAL METHOD FOR IDENTIFICATION OF SPECIFIC CUTTING FORCE DURING MILLING

M. Janota^{1*}, P. Kolář¹, M. Sulitka¹

¹Czech Technical University in Prague, Research Center of Manufacturing Technology, Prague, Czech Republic

*Corresponding author; e-mail: m.janota@rcmt.cvut.cz

Abstract

Specific cutting force is a key parameter that is important for estimating cutting forces that occur during machining. This information is important for various applications. The most important application is estimation of the stability limit valid for the specific configuration of the machine tool, tool and workpiece. There are a number of procedures used to predict the specific cutting force through various preliminary tests. This paper focuses on an operational method during milling that allows estimation of the specific cutting force using direct information from the machine tool control system. The specific cutting force is calculated as the ratio between the material removal rate and the power measured on the spindle. The method enables easy in-process identification of the specific cutting force that is valid for the specific workpiece material and the specific cutting edge geometry. The method is demonstrated on practical examples.

Keywords:

Specific cutting force; Mechanistic approach; Chatter; Frequency response function

1 INTRODUCTION

One of the obstacles to utilization of the installed power of a machine tool spindle is the regenerative vibration of the machine tool-tool-workpiece system, which is called chatter. Chatter is related to the cutting process and the interaction in the machine-tool-workpiece structure, as presented by Tlustý and Poláček [Tlustý 1957] and Tobias and Fishwick [Tobias 1958]. The regenerative vibrations depend on the chip thickness, the system's structural stiffness and the specific cutting force. This fact was taken into account in the first basic formula for milling applications, presented in [Tlustý 1957]:

$$a_{plim} = \frac{-1}{2 \cdot Z^* \cdot K_c \cdot \min(Re(FRF))} \quad (1)$$

Chatter simulation methods were investigated in subsequent years. Recent overviews of the state of the art of chatter research and chatter suppression were presented in [Altintas 2004], [Brecher 2009] and [Munoa 2016]. The tangential specific cutting force remains the fundamental parameter for chatter prediction characterizing the cutting ability of the cutting tool with specific cutting edge geometry during machining of a specific workpiece material.

Cutting force simulation is a cornerstone of chatter prediction. In general, the two main approaches are FEM-based and mechanistic models. FE methods also have the potential to simulate other related effects, typically the cutting temperature and tool wear. However, these methods need correctly identified fundamental material parameters [Arrazola 2013]. These FEM simulations are also time-demanding and therefore are not typically used to estimate the cutting force coefficients for chatter prediction.

Mechanistic models enable quick cutting force computation. These models are based on Martellotti's idea [Martellotti 1941, Martellotti 1945] that the cutting force is proportional to the uncut chip thickness and the specific cutting force (also called the cutting force coefficient). The specific cutting force can be modelled using an exponential model proposed by [Kienzle 1952] (2) or a linear model including the ploughing effect proposed by [Armarego 1969] (3):

$$F_c = k_c \cdot b \cdot h = k_{c1.1} \cdot h^{-m_c} \cdot K \cdot b \cdot h \quad (2)$$

$$F_c = k_{cc} \cdot b \cdot h + k_{ce} \cdot b \quad (3)$$

[Sabberwal 1962] presented a method for identification of the tangential, radial and axial cutting force coefficients related to the axial depth of cut and feed per tooth. [Fu 1984] and [Spiewak 1995] presented mechanistic models for identification of the cutting force coefficients using sets of experimental data. [Budak 1996] introduced an orthogonal to oblique transformation method that enables reduction of the number of experiments for the cutting force coefficients. Today, the orthogonal cutting database is recommended for solid end mills, drills, tools with a smooth rake face and a cutting edge without chamfer. Orthogonal database data can be successfully used for tools with a complex contour, e.g. helical tapered end mills or tools with a serrated cutting edge. On the contrary, mechanistic models based on a set of experimental data are recommended for exchangeable tips with a complex cutting edge geometry involving chip breakers, a varying rake surface and edge chamfer [Altintas 2012].

Specific cutting forces can be experimentally measured using two approaches: 1) direct measurement of cutting

forces using a stationary or rotary dynamometer and consequential computation using the uncut chip thickness and axial depth of cut; or 2) an indirect approach using spindle power monitoring during the machining operation and consequential computation using the tool diameter, uncut chip thickness and axial depth of cut.

When determining the tangential cutting force coefficient for milling using the direct measurement method, only one insert is clamped onto the tool to evaluate the cutting force. Jayram tried to overcome this shortcoming [Jayram 2001]. [Satyanarayana 2011] attempted to describe the relationship between the cutting force and plate geometry. The dependence of the tangential cutting force coefficient on cutting conditions was presented by [Velchev 2009] and [Karpuschewski 2018].

These experiments show that the cutting force coefficients depend on many different parameters related to the cutting process setting. The main advantage of indirect methods is very quick in-process identification of the cutting forces. [Qiu 2018] published a method using monitoring of the spindle and feed drive performance during machining and idle time for turning applications. A complex approach using monitoring of all drives during milling was presented by [Altintas 2017] for virtual monitoring of machining forces.

The last alternative for determining the cutting force coefficients are tool producer catalogue values (used most often in practice). These catalogues typically present coefficients for the Kienzle model; see e.g. [Mitsubishi 2019] or [Sandvik 2019].

This paper presents an operational method for identification of the tangential cutting force coefficient during milling for chatter prediction. The main motivation for developing this method for estimating the specific cutting force is potential future automated optimization of the cutting process without the need for any complex apparatus. The method is based on measurement of the spindle power during milling of a constant material volume with a constant feed rate. The anticipated main advantage is easy and quick prediction of the machining stability limit. The method is described in section 2. The experimental conditional and obtained data for two demonstration cases are presented in section 3. These data are analyzed using the proposed method and the standard mechanistic approach and are compared with catalogue values in section 4. An application for chatter limit prediction is also presented. Section 5 deals with the method uncertainties. The paper conclusion is presented in section 6.

2 DESCRIPTION OF PROPOSED METHOD

The machining performance of a machine tool is characterized by the volume of removed material per time unit (Material Removal Rate - MRR):

$$MRR = a_e \cdot a_p \cdot v_f \left[\frac{cm^3}{min} \right] \quad (5)$$

The MRR can be limited by the dynamic compliance of the structure or the installed spindle power. The relative performance calculated as the ratio of removed material and installed spindle power $MRR/P \left[\frac{cm^3/min}{kW} \right]$ characterizes the force interaction between the tool cutting edge and the workpiece material. The MRR/P ratio is a reciprocal value of the tangential cutting force coefficient, as the basic unit analysis shows:

$$\left[\frac{MRR}{P} \right] = \left[\frac{1}{K_c} \right] \quad (6a)$$

$$\left[\frac{\left[\frac{m^3}{s} \right]}{[W]} \right] = \left[\frac{\left[\frac{m^3}{s} \right]}{\left[\frac{Nm}{s} \right]} \right] = \left[\frac{m^2}{N} \right] = \frac{1}{[Pa]} \quad (6b)$$

P is the spindle power consumed with the cutting process; MRR is the metal removal rate during constant immersion milling (5). The spindle power P is characterized as the product of the spindle rotational speed and its torque. The power P is concurrently the product of the tangential cutting force and cutting speed. Thus, K_c in (6a) is the tangential cutting force coefficient that could also be used for chatter prediction simulation (1).

The proposed method is based on identification of K_c through direct measurement of the spindle power during constant immersion milling and subsequent calculation of the cutting force coefficient. The spindle power and the feed can be measured as the waveforms of time-controlled axes: Servo trace (Sinumerik), TNCscope (HEIDENHAIN) and Servo Guide (Fanuc) respectively. The power consumed with the cutting process would be calculated as the difference between the measured total spindle power P_t and the spindle power during idle running P_0 :

$$P = P_t - P_0 \quad (7)$$

The tangential cutting force coefficient can be calculated directly:

$$K_c = \frac{P}{MRR} \quad (8)$$

3 REFERENCE IDENTIFICATION OF THE SPECIFIC CUTTING FORCE

3.1 Experiment setup

Two experiments of C45 steel milling were conducted to verify the proposed method. Two different cutting tools on two different machine tools were used; see Tab. 1 and Tab. 2. There was only one insert clamped onto the tool body for the basic experiments to enable clear analysis of the dynamometer signal. Another setup with more mounted cutting inserts was tested later (see section 4.2). The workpiece material was placed on a three-axis KISTLER 9255B dynamometer and a KISTLER 5017B1500 amplifier in order to obtain the reference data. The amplified data was sampled and processed using a B&K PULSE analyzer. The experimental setup is shown in Fig. 1.

The actual measurement procedure was as follows. First, the linearity of the FRF dynamometer was verified using a modal hammer; see Fig. 2. The tool body was equipped with one insert only. Thus, the maximum tooth pass frequency was about 25.3 Hz. For each measurement case, three cutting force components were measured: F_x , F_y and F_z . Concurrently, the spindle power was measured in the machine tool control system directly. There was one run of the experiment for every combination $f_z - v_c$. Five levels of the cutting speed v_c were used for every feed per tooth value f_z . The experiment was not full factorial; see the detailed overview in Appendix 1 for more details. These measured force data were used for subsequent analysis of the cutting coefficients; see the next section. The data obtained from the dynamometer were used to calculate the reference value of the cutting force coefficients.

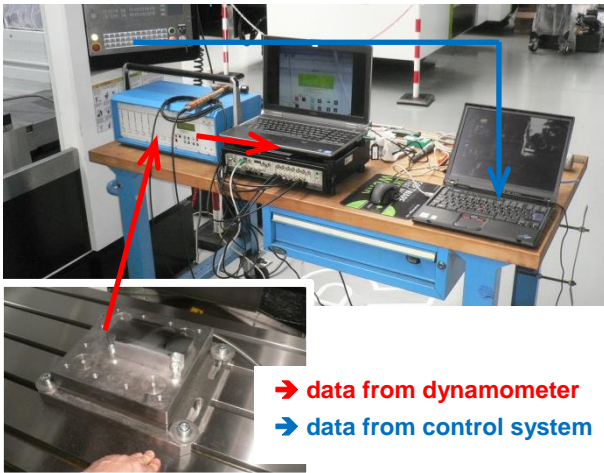


Fig. 1: Measurement apparatus.

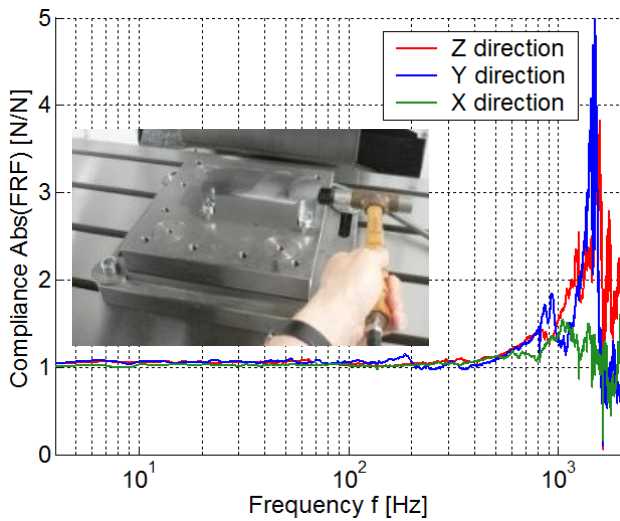


Fig. 2: Frequency response function of dynamometer.

Tab. 1: Case study 1: Tool and test conditions.

Machined material:	C45 (170 HB) 1.1191 (W.Nr.)
Tool body:	TGS F1600.100.N32.50.14.Z7.C Dc = 100 mm z = 7
Insert:	SUMITOMO AXMT170508PEERH ACP200
Cutting conditions:	$v_c = 100 150 200 250 300$ m/min $f_z = 0.10 0.15 0.20 0.25 0.30$ mm $a_e = 82$ mm $a_p = 1$ mm Face milling
Machine tool:	Three-axis vertical machining centre Belt driven spindle 15 kW/8000 rpm HEIDENHAIN TNC620 control system

Tab. 2: Case study 2: Tool and test conditions.

Machined material:	C45 (170 HB) 1.1191 (W.Nr.)
Tool body:	ISCAR 3M F90AX D063-27-20 Dc = 63 mm z = 5
Insert:	ISCAR 3M AXKT 2006PDTR-RM
Cutting conditions:	$v_c = 100/150/200/250/300$ m/min $f_z = 0.10/0.15/0.20/0.25/0.30$ mm $a_e = 63$ mm $a_p = 1$ mm Slot milling
Machine tool	Five-axis vertical machining centre with rotary-tilting table Belt driven spindle 40 kW/10000 rpm HEIDENHAIN TNC620 control system

3.2 Specific cutting force calculation from the dynamometer data

The procedure for processing the measured data was as follows. In the first step, three cutting force components were measured using the dynamometer for each cutting condition in the stationary coordinate system of the machine tool: F_x , F_y , and F_z . Then, a low-pass filter was applied to all components to remove unwanted high frequency components (Fig. 3). The active force F_a was then calculated using Eq. 9:

$$F_a = \sqrt{F_x^2 + F_y^2} \quad (9)$$

The maximum F_a was identified. For this situation, the chip thickness was considered in a simplified way as $h = f_z$. The parameter K_c was then calculated using Eq. 10:

$$K_c = \frac{\max(F_a)}{f_z \cdot a_p} \quad (10)$$

Note that Eq. 10 is valid for clearly separated signals of every engaged tooth. In this case, a one-tooth tool was used, thus this basic condition was satisfied.

The resulting dependence of K_c on the feed per tooth f_z is shown in Fig. 5. Kienzle model coefficients using least-square regression were computed to enable a comparison with typical catalogue values:

$$K_c(f_z) = k_{c1.1} \cdot h^{-m_c} \quad (11)$$

For this specific case, $h = f_z$. The results of the regression are shown in Tab. 4A.

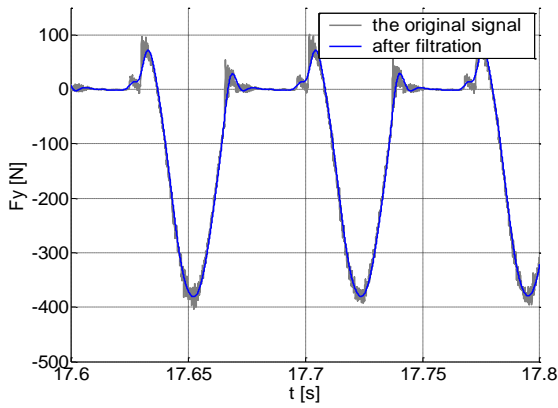


Fig. 3: Example of low-pass filtering of measured data.

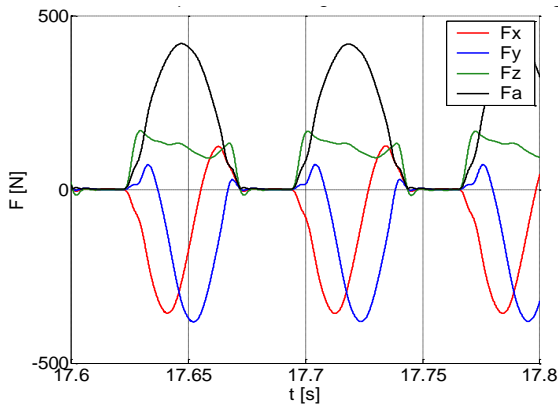


Fig. 4: Cutting forces F_x , F_y , F_z measured with the dynamometer and calculated active force F_a (9).

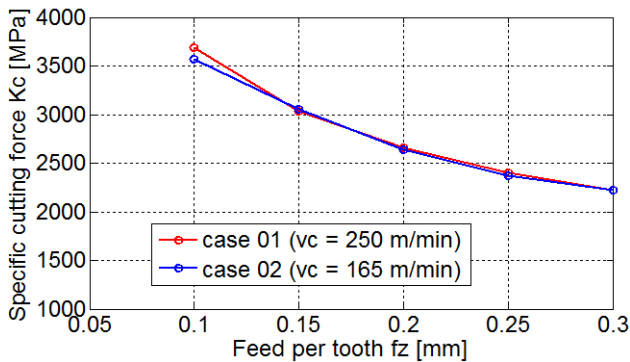


Fig. 5: Dependence of calculated specific cutting force K_c on feed per tooth f_z for both cases using the dynamometer data.

Looking at the dependence of the specific cutting force K_c on the feed per tooth f_z , it is clear that K_c decreases as f_z increases. This fact is well known. It confirms that the data measured with the dynamometer are correct. These data are used in the next section as the reference data for validation of the P/MRR method results.

4 IDENTIFICATION OF THE SPECIFIC CUTTING FORCE USING THE P/MRR METHOD

4.1 Procedure description

The spindle power consumption was recorded using the machine tool control system during idle running and during machining (Fig. 6). The power consumed with the cutting process was calculated:

$$P = \text{mean}(P_t) - \text{mean}(P_o) \quad (12)$$

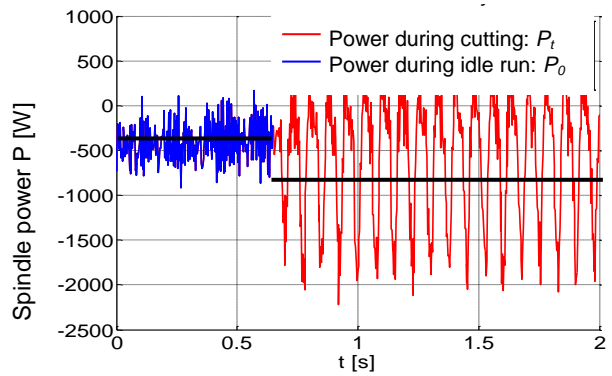


Fig. 6: Example of power signal measurement. The black lines indicate mean signal values.

The metal removal rate MRR was calculated using known immersion values and the feed rate (5). The specific cutting force K_c was calculated using Eq. 8. The results of machining with one installed insert were measured and evaluated for both tools. For case study 02, machining with the full number of installed inserts was performed.

4.2 Comparison of results

The dependence of the calculated specific cutting force K_c on the feed per tooth f_z is presented in Fig. 7 and Fig. 9. The characteristic curve shape is visible again. The results from the dynamometer are used as a reference. The deviation of the P/MRR results is about $\pm 5\%$ compared to the dynamometer results. The results calculated from the tool with one installed insert and five installed inserts are almost identical; see Fig. 9.

The data obtained from the Sandvik catalogue are presented as a typical example of the data available in the industry. For milling of C45 steel, the catalogue gives underestimated values for lower f_z values.

Kienzle model coefficients were identified for the P/MRR results and also for catalogue data using Eq. 11; see Tab. 4B and Tab. 4C.

The dependence of the calculated specific cutting force K_c on the cutting speed v_c is presented in Fig. 8 and Fig. 10. The K_c value has low sensitivity to the v_c as would be expected for the cutting speed range of 100 - 300 m/min.

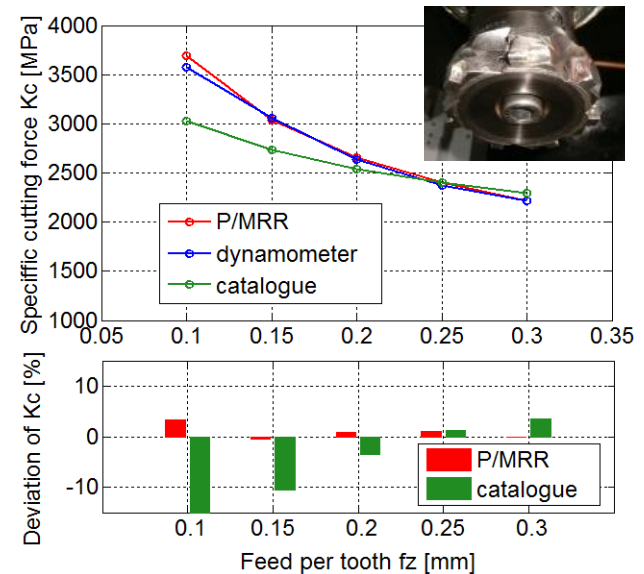


Fig. 7: Dependence of K_c on f_z for case study 01. Example for $v_c = 250$ m/min. The dynamometer data is a reference for the deviation calculation.

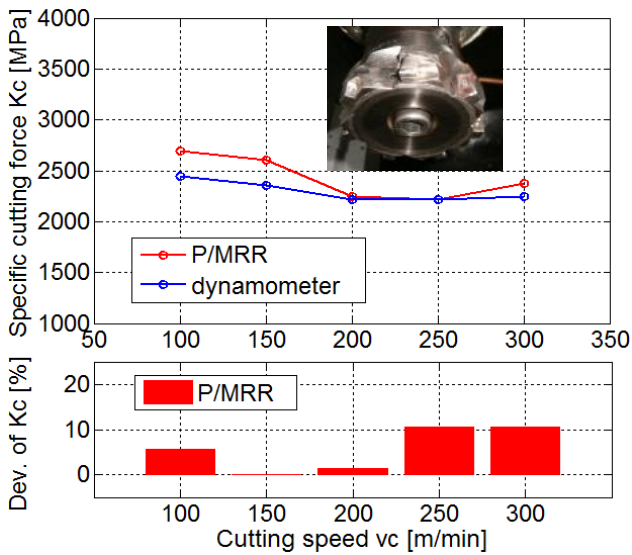


Fig. 8: Dependence of K_c on v_c for case study 01. Example for $f_z = 0.30$ mm. The dynamometer data is a reference for the deviation calculation.

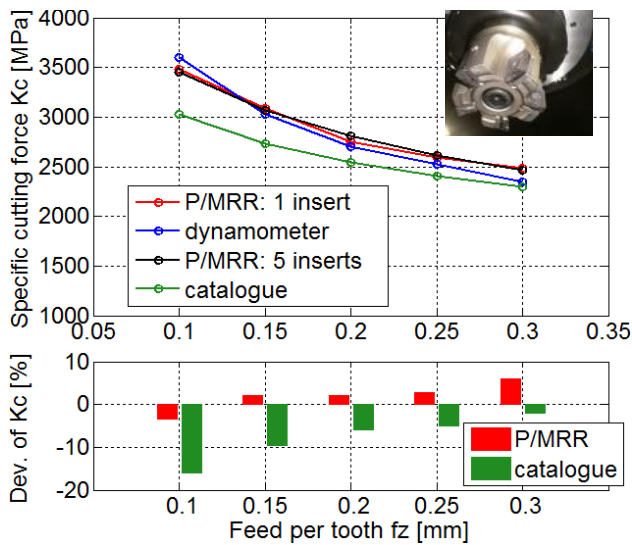


Fig. 9: Dependence of K_c on f_z for case study 02. Example for $v_c = 165$ m/min. The dynamometer data is a reference for the deviation calculation.

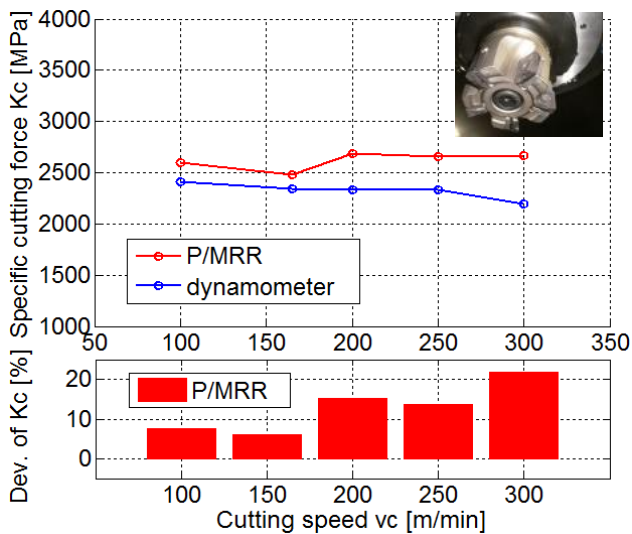


Fig. 10: Dependence of K_c on v_c for case study 02. Example for $f_z = 0.30$ mm. The dynamometer data is a reference for the deviation calculation.

Tab. 4: Cutting force coefficients of the Kienzle model identified from various data sources.

A) Data obtained from dynamometer measurement		
Case no.	$k_{c1.1}$ [MPa/mm]	m_c [-]
01	1294	0.44
02	1498	0.39

B) Data calculated using the P/MRR method		
Case no.	$k_{c1.1}$ [MPa/mm]	m_c [-]
01	1262	0.47
02 (1 insert)	1687	0.31
02 (5 inserts)	1713	0.31

C) Data calculated from tool catalogue figures		
Catalogue	$k_{c1.1}$ [MPa/mm]	m_c [-]
[Sandvik 2019]	1700	0.25

4.3 Verification with stability lobe diagram

An analysis of the stable machining conditions was conducted as an additional validation of the results. The specific cutting force K_c estimated with the P/MRR method was used for frequency domain simulation (1) of the machining [SchmitzSmith 2009] using the F1600 tool (case 01, $D = 100$ mm, $Z = 7$). As an input, the dynamic compliance was measured on the tool tip using a hammer tap test; see Fig. 11. The simulated stability limit was validated with cutting tests; see Fig. 12.

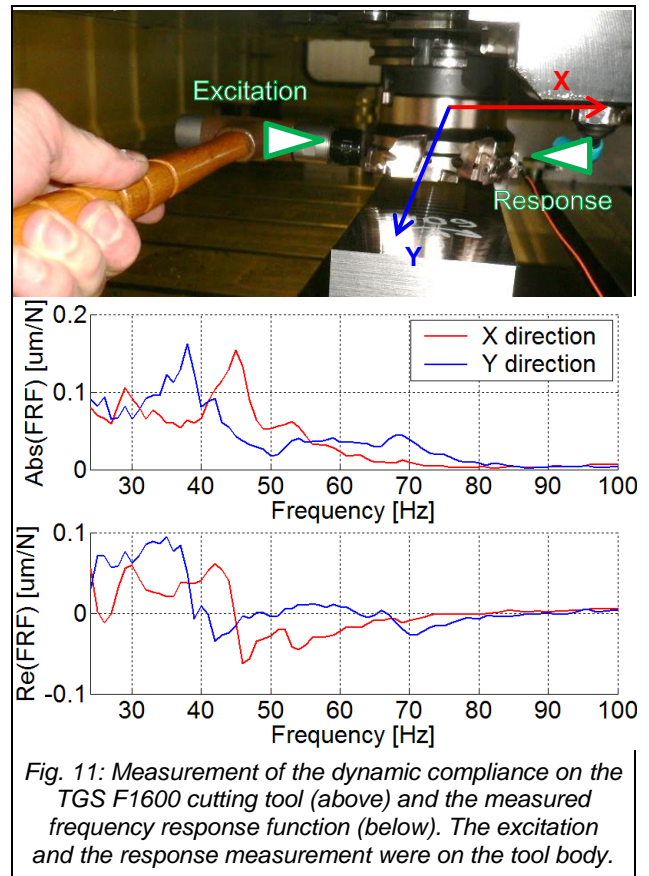


Fig. 11: Measurement of the dynamic compliance on the TGS F1600 cutting tool (above) and the measured frequency response function (below). The excitation and the response measurement were on the tool body.

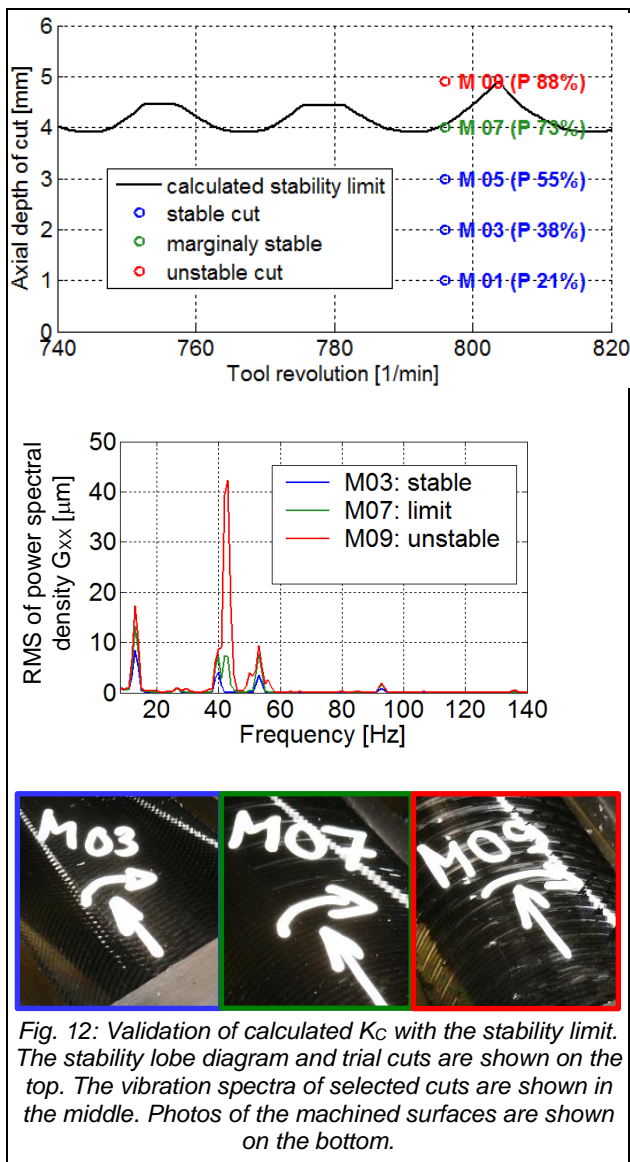


Fig. 12: Validation of calculated K_c with the stability limit. The stability lobe diagram and trial cuts are shown on the top. The vibration spectra of selected cuts are shown in the middle. Photos of the machined surfaces are shown on the bottom.

5 DISCUSSION

An operational method for estimation of the tangential specific cutting force was presented. The main advantage of the method is that it is a quick and easy procedure that enables identification of the tangential specific cutting force as an important characterization of the specific cutting process.

The comparison of one-tooth-machining data showed that the P/MRR method yields results as a traditional approach using the dynamometer with an acceptable deviation within the range of $\pm 5\%$. In general, the method uses the machining process total energy (12) and the total volume of the machined material. Thus, the number of currently engaged teeth is not important for the calculation as shown by the comparison of one-tooth-machining and five-teeth-machining; see Fig 9. The cutting force coefficient calculated as the ratio of two total values also provides an opportunity to minimize the importance of other effects such as oscillation of the spindle power or local material inhomogeneity.

The presented machining experiments were performed with a low axial depth of cut of 1 mm to ensure a stable cut. In this case, the measurement results could be affected by the size of the cutting edge radius and the tool tip radius. It is probable that the real influence of these parameters is not

very strong. The estimated specific cutting force was used for relevant prediction of the machining stability limit even on higher depths of cut (about $a_p = 4$ mm). Nevertheless, further work is needed in this context to estimate the sensitivity of the method on tool engagement parameters.

Sumitomo and Iscar inserts were used. The results of calculated K_c were compared with the Sandvik catalogue values, which has highly specified materials. The maximum difference of the catalogue values from the dynamometer results was -15% . The error is lower for higher feed per tooth values. Thus, the general data for C45 steel machining available in the catalogue are also useful.

The proposed method enables identification of the tangential specific cutting force k_c during machining without the need to install any special force sensors. In industrial applications, continuous measurement would not be easy due to missing information about the real removed material volume per time (MRR). This information could be provided as a specific data file generated e.g. by simulation software that is able to simulate the removed material volume. As an alternative simplified method, the measurement could be done only at selected tool path sections where the removed material volume is well known.

6 SUMMARY

An operational method for specific cutting force estimation was presented. The method is based on calculation of the P/MRR ratio. Two use cases executed on various two machine tools and two various tools show that the P/MRR method results have a low difference of $\pm 5\%$ compared to the reference values measured with the dynamometer. In addition, the estimated specific cutting force value was successfully used to predict the machining stability limit. The entire procedure could be automated easily. One specialized cycle for making the identification cut can provide useful data for analysis of the cutting tool conditions.

7 ACKNOWLEDGMENTS

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NOMENCLATURE:

F_x, F_y, F_z	[N]	cutting forces in the stationary coordinate system of the machine tool
F_A	[N]	active cutting force
F_C	[N]	tangential cutting force
G_{xx}	[m]	RMS value of power spectral density
K	[-]	total correction factor
K_C	[MPa]	specific cutting force
MMR	[cm ³ /min]	metal removal rate
P	[kW]	power consumed by the cutting process
P_t	[kW]	total spindle power measured during machining
P_0	[kW]	spindle power during idle running
Z	[-]	number of the tool teeth
Z^*	[-]	teeth number engaged in the workpiece
a_e	[mm]	radial depth of cut
a_p	[mm]	axial depth of cut
f_z	[mm]	feed per tooth
f_v	[mm/min]	feed rate
k_{C11}	[MPa]	specific cutting force, $a_p=h=1$ mm
h	[mm]	average chip thickness
m_C	[-]	exponent of the specific cutting force
v_C	[m/min]	cutting speed

APPENDIX 1: OVERVIEW OF THE MEASURED FIGURES

Table 5: Experiment data for case study 01, machining with one tooth.

ID	a_p [mm]	a_e [mm]	V_c [m/min]	f_z [mm]	Z [-]	MRR [cm ³ /min]	P_o [kW]	P_t [kW]	K_c [MPa]*
1	1	82	250	0.10	1	6.5	0.27	0.68	3689.9
2	1	82	250	0.15	1	9.8	0.27	0.77	3038.5
3	1	82	250	0.20	1	13.1	0.27	0.85	2657.0
4	1	82	250	0.25	1	16.3	0.27	0.93	2398.5
5	1	82	250	0.30	1	19.6	0.27	1.00	2217.9
6	1	82	200	0.30	1	15.7	0.25	0.84	2251.6
7	1	82	150	0.30	1	11.7	0.34	0.85	2603.1
8	1	82	100	0.30	1	7.8	0.19	0.54	2696.6
9	1	82	300	0.30	1	23.5	0.32	1.25	2376.3

Table 6: Experiment data for case study 02, machining with 1 tooth and 5 teeth.

ID	a_p [mm]	a_e [mm]	V_c [m/min]	f_z [mm]	Z [-]	MRR [cm ³ /min]	P_o [kW]	P_t [kW]	K_c [MPa]*
10	1	63	165	0.10	1	5.3	0.14	0.44	3477.6
11	1	63	165	0.15	1	7.9	0.14	0.54	3084.9
12	1	63	165	0.20	1	10.5	0.14	0.62	2753.4
13	1	63	165	0.25	1	13.1	0.14	0.71	2596.7
14	1	63	165	0.30	1	15.8	0.14	0.79	2485.0
15	1	63	165	0.10	1	5.3	0.14	0.48	3936.5
16	1	63	100	0.30	1	9.5	0.09	0.50	2597.4
17	1	63	200	0.30	1	19.1	0.14	0.99	2684.8
18	1	63	250	0.30	1	23.9	0.10	1.15	2655.0
19	1	63	300	0.30	1	28.6	0.18	1.45	2671.8
20	1	63	165	0.20	1	10.5	0.14	0.66	2958.8
21	1	63	165	0.25	1	13.1	0.14	0.73	2686.7
22	1	63	165	0.10	5	26.3	0.14	1.65	3451.8
23	1	63	165	0.15	5	39.4	0.14	2.15	3068.2
24	1	63	165	0.20	5	52.5	0.14	2.60	2815.7
25	1	63	165	0.25	5	65.7	0.14	3.00	2609.9
26	1	63	165	0.30	5	78.8	0.14	3.38	2469.8
27	1	63	165	0.20	5	52.5	0.14	2.60	2806.9

* K_c calculated using the P/MRR method